





wherein  $x$ ,  $y$  and  $z$  locate the sites at which the temperature is calculated vs.  $\xi$ ,  $\eta$ , and  $\zeta$  which locate the heat sources. Once the heat source has been located at  $\xi = 0, \eta = 0, \zeta = 0$  (i.e. at the coordinate origin) the  $\xi, \eta, \zeta$  coordinates need not be mentioned further. Both  $\xi, \eta, \zeta$  and  $x, y, z$  coordinates move together.

The component terms in equation (13) diminish with  $n$  and permit  $T - T_o$  to converge and to be evaluated without problems by computer.

For complete joint penetration welds where the weld root and face widths are identical, the temperature distribution of equation (13) approaches a moving line source:

$$T - T_o = \frac{P}{2\pi kd} e^{-\frac{v}{2\alpha}x} K_o\left(\frac{v}{2\alpha}R\right) \quad (16)$$

where  $K_o$  is the zeroth order modified Bessel function of the third kind and:

$$R = \sqrt{x^2 + y^2} \quad (17)$$

### Dipole Phase Change Representation

As a weld pool moves forward, the forward surface melts and the rear surface freezes. Under steady state conditions the temperature perturbation  $\Delta T$  produced by the phase change should be

$$\Delta T = \oint_{A'} \rho L v U(\xi, \eta, \zeta) d\eta d\zeta \quad (18)$$

where  $\rho$  and  $L$  are the density and latent heat of fusion respectively for the weld metal.

When the integral is evaluated, it is necessary to add a correction which cancels out the heat flow induced through the supposedly insulating half space surface from the latent heat sources on the cooling and freezing sites on the molten weld pool surface. This is accomplished by placing a symmetrical image array of latent heat sources (i.e., a symmetrical molten weld pool interface) above the half space surface.

Integration over this symmetrical, closed surface of the solid-liquid molten weld pool interface in an infinite, unbounded continuum allows symmetry for a nonconducting plane at the half space boundary; also, it adds no additional heat, since symmetry also distributes the heat from the two sets of heat sources into two symmetrical half spaces. Thus,  $A'$ , the surface of integration, should be taken to include the symmetrical "image surface" above the boundary plane of the half space containing the molten weld pool.

Representing the above heat source distribution by a pair of equal and opposite monopoles as shown in Fig. 2 of

strength  $2P$  spaced at  $+\frac{\Delta \xi}{2}$  and  $-\frac{\Delta \xi}{2}$  on the  $\eta = 0, \zeta = 0$  axis required

by radial symmetry:

$$\Delta T \approx 2P \left[ U\left(\frac{\Delta \xi}{2}\right) - U\left(-\frac{\Delta \xi}{2}\right) \right] \quad (19)$$

or

$$\Delta T \approx 2P \Delta \xi \left[ \frac{U\left(\frac{\Delta \xi}{2}\right) - U\left(-\frac{\Delta \xi}{2}\right)}{\Delta \xi} \right] \quad (20)$$

If  $P$  is allowed to become very large as  $\Delta \xi$  approaches zero so that a limit  $\dot{Q}_\xi$  exists such that:

$$\dot{Q}_\xi = \lim_{\Delta \xi \rightarrow 0} 2P \Delta \xi \quad (21)$$

then as  $\Delta \xi$  approaches zero,

$$\Delta T \approx \dot{Q}_\xi \left( \frac{\partial U}{\partial \xi} \right)_o \quad (22)$$

where:

$$\left( \frac{\partial U}{\partial \xi} \right)_o = U_o \left[ \left( 1 + \frac{x}{r} \right) \frac{v}{2\alpha} + \frac{x}{r^2} \right] \quad (23)$$

The proof of equation (23) proceeds as follows from equation (4):

$$U = \frac{1}{4\pi k} \frac{e^{-\frac{v}{2\alpha}(r+x-\xi)}}{r} \quad (4)$$

$$\frac{\partial U}{\partial \xi} = \frac{1}{4\pi k} \left[ e^{-\frac{v}{2\alpha}(r+x-\xi)} \frac{\partial}{\partial \xi} \left( \frac{1}{r} \right) + \frac{1}{r} \frac{\partial}{\partial \xi} \left( e^{-\frac{v}{2\alpha}(r+x-\xi)} \right) \right] \quad (1a)$$

$$= \frac{1}{4\pi k} \left[ e^{-\frac{v}{2\alpha}(r+x-\xi)} \left( -\frac{1}{r^2} \frac{\partial r}{\partial \xi} \right) + \frac{1}{r} \left( e^{-\frac{v}{2\alpha}(r+x-\xi)} \left( -\frac{v}{2\alpha} \right) \left( \frac{\partial r}{\partial \xi} - 1 \right) \right) \right] \quad (2a)$$

$$\begin{aligned} \frac{\partial r}{\partial \xi} &= \frac{\partial}{\partial \xi} \sqrt{(x-\xi)^2 + (y-\eta)^2 + (z-\zeta)^2} \\ &= \frac{1}{2} \frac{2(x-\xi)(-1)}{\sqrt{(x-\xi)^2 + (y-\eta)^2 + (z-\zeta)^2}} \\ &= -\frac{x-\xi}{r} \end{aligned} \quad (3a)$$





or:

$$\Delta T = 2P\Delta\xi^2 \left\{ \frac{\left[ \frac{U(\xi + \Delta\xi) - U(\xi)}{\Delta\xi} \right] - \left[ \frac{U(\xi) - U(\xi - \Delta\xi)}{\Delta\xi} \right]}{\Delta\xi} \right\} \quad (25)$$

If P is allowed to become very large as  $\Delta\xi$  approaches zero so that a limit  $\dot{Q}_{\xi\xi}$  exists such that

$$\dot{Q}_{\xi\xi} = \lim_{\Delta\xi \rightarrow 0} 2P(\Delta\xi)^2 \quad (26)$$

then, as  $\Delta\xi$  approaches zero:

$$\Delta T \approx \dot{Q}_{\xi\xi} \left( \frac{\partial^2 U}{\partial \xi^2} \right)_o \quad (27)$$

Similarly:

$$\dot{Q}_{\eta\eta} = \lim_{\Delta\eta \rightarrow 0} 2P(\Delta\eta)^2 \quad (28)$$

However, since the source displacements in the z-direction which create the quadrupole  $\dot{Q}_{\xi\xi}$  need no additional image force, it is more appropriate to write:

$$\dot{Q}_{\xi\xi} = \lim_{\Delta\xi \rightarrow 0} P(\Delta\xi)^2. \quad (29)$$

The nature of the circulations is such that  $\dot{Q}_{\xi\xi}$  should be of opposite sign to  $\dot{Q}_{\xi\xi}$  and  $\dot{Q}_{\eta\eta}$ . This is to say that, when heat is flowing outwards (positive  $\dot{Q}_{\xi\xi}$  and  $\dot{Q}_{\eta\eta}$ ) along the top of the weld pool due to fluid transport, the upward flow of fluid from the bottom of the pool superimposes a component of heat flow away from the pool bottom towards the surface (negative  $\dot{Q}_{\xi\xi}$ ) onto the original heat flow pattern. The total contribution of heat flow patterns around a weld pool due to internal fluid circulations is then:

$$\Delta T = \dot{Q}_{\xi\xi} \left( \frac{\partial^2 U}{\partial \xi^2} \right)_o + \dot{Q}_{\eta\eta} \left( \frac{\partial^2 U}{\partial \eta^2} \right)_o + \dot{Q}_{\xi\xi} \left( \frac{\partial^2 U}{\partial \xi^2} \right)_o \quad (30)$$

Where  $\dot{Q}_{\xi\xi}$ ,  $\dot{Q}_{\eta\eta}$ , and  $\dot{Q}_{\xi\xi}$  are independent variables (except for the above mentioned loose relation), and:

$$\begin{aligned} \left( \frac{\partial^2 U}{\partial \xi^2} \right)_o &= U_o \left[ \left( 1 + 2\frac{x}{r} + \frac{x^2}{r^2} \right) \left( \frac{V}{2\alpha} \right)^2 \right. \\ &\quad \left. + \left( -1 + 2\frac{x}{r} + 3\frac{x^2}{r^2} \right) \left( \frac{V}{2\alpha} \right) \left( \frac{1}{r} \right) \right. \\ &\quad \left. + \left( -1 + 3\frac{x^2}{r^2} \right) \left( \frac{1}{r} \right)^2 \right] \quad (31) \end{aligned}$$

$$\begin{aligned} \left( \frac{\partial^2 U}{\partial \eta^2} \right)_o &= U_o \left[ \left( \frac{y^2}{r^2} \right) \left( \frac{V}{2\alpha} \right)^2 + \left( -1 + 3\frac{y^2}{r^2} \right) \left( \frac{V}{2\alpha} \right) \left( \frac{1}{r} \right) \right. \\ &\quad \left. + \left( -1 + 3\frac{y^2}{r^2} \right) \left( \frac{1}{r} \right)^2 \right] \quad (32) \end{aligned}$$

$$\left(\frac{\partial^2 U}{\partial z^2}\right)_0 = U_0 \left[ \left(\frac{z^2}{r^2}\right) \left(\frac{V}{2\alpha}\right)^2 + \left(-1 + 3\frac{z^2}{r^2}\right) \left(\frac{V}{2\alpha}\right) \left(\frac{1}{r}\right) + \left(-1 + 3\frac{z^2}{r^2}\right) \left(\frac{1}{r}\right)^2 \right] \quad (33)$$

### Conclusion

For a weld made by a heat source of effective power  $P$ , i.e., the power supplied by the welding unit multiplied by a process efficiency factor, moving with constant velocity  $V$  in the  $x$ -direction on the surface ( $x$ - $y$  plane) of a wide, long plate of thickness  $d$ , the temperature field  $T$  may be represented approximately by the expression:

$$T = T_0 + P \left[ F(n=0) + \sum_{n=1}^{\infty} (F(r_n) + F(r'_n)) \right] \quad (34)$$

where  $T_0$  = plate base temperature;  $r_n = \sqrt{x^2 + y^2 + (z - 2nd)^2}$ , (equation 14);  $r'_n = \sqrt{x^2 + y^2 + (z + 2nd)^2}$ , (equation 15). and:

$$F(r) = \frac{1}{2\pi k} \frac{e^{-V(r+x)/2\alpha}}{r} \left\{ 1 + \left(\frac{Q_{\xi}}{P}\right) \left[ \left(1 + \frac{x}{r}\right) \frac{V}{2\alpha} + \left(\frac{x}{r}\right) \left(\frac{1}{r}\right) + \left(\frac{Q_{\xi\xi}}{P}\right) \left[ \left(1 + 2\frac{x}{r} + \left(\frac{x}{r}\right)^2\right) \left(\frac{V}{2\alpha}\right)^2 + \left(-1 + 2\frac{x}{r} + 3\left(\frac{x}{r}\right)^2\right) \left(\frac{V}{2\alpha}\right) \left(\frac{1}{r}\right) + \left(-1 + 3\left(\frac{x}{r}\right)^2\right) \left(\frac{1}{r}\right)^2 \right] + \left(\frac{Q_{\eta\eta}}{P}\right) \left[ \left(\frac{y}{r}\right)^2 \left(\frac{V}{2\alpha}\right)^2 + \left(-1 + 3\left(\frac{y}{r}\right)^2\right) \left(\frac{V}{2\alpha}\right) \left(\frac{1}{r}\right) + \left(-1 + 3\left(\frac{y}{r}\right)^2\right) \left(\frac{1}{r}\right)^2 \right] + \left(\frac{Q_{\zeta\zeta}}{P}\right) \left[ \left(\frac{z}{r}\right)^2 \left(\frac{V}{2\alpha}\right)^2 + \left(-1 + 3\left(\frac{z}{r}\right)^2\right) \left(\frac{V}{2\alpha}\right) \left(\frac{1}{r}\right) + \left(-1 + 3\left(\frac{z}{r}\right)^2\right) \left(\frac{1}{r}\right)^2 \right] \right\} \quad (35)$$

$Q_{\xi}$  is a thermal dipole representing the phase change contribution to the temperature field.  $Q_{\xi}/P$  is a length which would be expected to be negative and on the order of magnitude of the size of the weld puddle.

$Q_{\xi\xi}$ ,  $Q_{\eta\eta}$ , and  $Q_{\zeta\zeta}$  are quadrupole moments representing the contribution of fluid circulations in the weld puddle to the surrounding temperature distribution. For circulations upwards and out at the weld center the first two terms should be positive and the third, negative. The terms  $Q_{\xi\xi}/P$ ,  $Q_{\eta\eta}/P$ , and  $Q_{\zeta\zeta}/P$  have the dimensions of the square of a length. Wide variations in these terms may be expected depending upon widely different flow conditions in the weld pool.

Weld models employing more variables must, surely, offer possibilities for better fits to empirical data. Variables with physical significance yield insight into physical phenomena and thus are of special value. If a best fit of the above model to empirical data consistently yields process efficiencies and multipole values within expectation bands provided by theory and other kinds of empirical data, then indications will be that the model does provide the physical information which it is supposed to provide. In the meantime the model is proposed as a tentative approach to obtaining insight into heat flow in one particular kind of weld.

### Acknowledgment

This work was carried out at NASA's Marshall Space Flight Center and is published by permission (MSFC Form 2818).

### References

1. Rosenthal, D. 1935. Étude théorique du régime thermique pendant la soudure à l'arc. *Congrès national des sciences*, 2d, Brussels: 1277-1292.
2. Rosenthal, D., and Schmerber, R. 1938. Thermal study of arc welding. *Welding Journal* 17(4):2-8.
3. Rosenthal, D. 1941. Mathematical theory of heat distribution during welding and cutting. *Welding Journal* 20(5):220-s to 234-s.
4. Rosenthal, D. 1946 (Nov.). The theory of moving sources of heat and its application to metal treatments. *Transactions of the American Society of Mechanical Engineers* 68:849-866.
5. Ishizaki, K., Murai, K., and Kanbe, Y. 1966 (April). *Penetration in arc welding and convection in molten metal*. International Institute of Welding Study Group 212, document 77-66.
6. Mills, G. S. 1979. Fundamental mechanisms of penetration in GTA welding. *Welding Journal* 58(1):21-s to 24-s.
7. Heiple, C. R., and Roper, J. R. 1982. Mechanism for minor element effect on GTA fusion zone geometry. *Welding Journal* 61(4):97-s to 102-s.
8. Apps, R. L., and Milner, D. R. 1955. Heat flow in argon-arc welding. *British Welding Journal* 2(10):475-485.